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Experimental Analysis of Aerodynamic Forces for Cross Flow on Single Rectangular Cylinder

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ABSTRACT

An experimental investigation of static pressure distributions on rectangular cylinders for a uniform cross flow is performed. The effect of side ratios and angle of attack are encountered in the investigation. Drag and lift coefficients are presented which are calculated from measured pressure distributions.

NOMENCLATURE

A	Area
CD	Drag coefficient
CL	Lift coefficient
Cp	Pressure coefficient
D	Width of Cylinder
FD	Drag Force
FL	Lift Force
H	Breadth of Cylinder
TIM	C'1- D.1' CO 1'- 1

- H/D Side Ratio of Cylinder
- L₁ Longitudinal Spacing
- Lt Transverse Spacing
- P Local Static Pressure
- Po Free Stream Static Pressure
- U_o Free Stream Velocity
- α Angle of Attack
- o Density of air

INTRODUCTION

Engineering problems regarding wind loads around skyscrapers, chimneys, towers and the flow induced vibrations of tubes in heat exchangers, bridges, oil rigs or marine structures need detailed investigation of flow patterns and aerodynamic characteristics on bluff bodies. In the case of flow over a bluff body the separation points are fixed at the leading edges and the shear layers originating from the leading corners curve outwards and a wake region is formed behind the body.

Till now extensive research work has been carried out on flow over isolated bluff bodies. P.W.Bearman and D.M.Trueman (1972) investigated the base pressure coefficient, drag coefficient and Strouhal number of rectangular cylinders. Y.Nakamura and Yujiohya (1986) attempted to study vortex shedding from square prisms placed normal to smooth and turbulent approach flows. B.E.Lee(1975) made an elaborate study of the effect of turbulence on the surface pressure field of a square prism. A.R.Barriga, C.T.Crowe and J.A.Roberson (1975) studied the effects of angle of attack, turbulence intensity and scale on the pressure distribution of a single cylinder.

Rectangular cylinders ideally represent the general shape of tall buildings. So wind tunnel uxperiments on rectangular cylinders would be useful in the analysis of wind effects on tall buildings. The present experimental investigation is an attempt to make an understanding of the nature of mean pressure distributions and drag on rectangular cylinders with varying side ratios (ratio of breadth H to width D of the cross section of cylinder). The experiment is conducted for a uniform cross flow. The flow direction is varied in order to determine the effect of angle of attack on pressure distribution. The present experimental results are compared with the existing ones.

EXPERIMENTAL SET UP AND PRODCEDURE

The experiment was carried out in an open circuit subsonic wind tunnel with a test section of 457mm X 457mm (18 inch X 18 inch) cross section. Four rectangular cylinders were used. The cylinders spanned 457mm each and their side dimensions were: width D=30mm for each, breadth H=37.5mm, 45mm, 52.5mm and 60mm. Each rectangular cylinder was tapped on two adjacent sides to measure pressure distribution. Flexible tubes of 1.6mm outer diameter were used to connect the tappings to the limbs of a multimanometer. Water was used as the manometric fluid.

The rectangular cylinder of side ratio H/D=1.25

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was mounted centrally gr a foreigntal plane in the test section so that the Komm Late was original normal to the flow direction free of . For angular orientation of the cylinders a graduated theorems used. Mean presente distribution was recorded at angles of attack varying from 6-10 ×5° with astep of 5°.

Pressure distributions for each of sic cylinders events and cratto of d 25, 1.2, 1.75 and 2.0 were meetinged in a similar manner.

The flow velocity in the test section was kept constraint at 18.3 m/s (60fps). The Rejectida dimitian based on the side dimension D=30mm was \$3,45,10⁴, the subtlence intensity of the tunnel was



– Wooden side wall

End view of test section



the back surface of cylinder at angles of attack to 0° 10°, 20°, 30° and 45° are shown in figure 7.11 is seen that at 0° angle of attack (figure 7a) minimum Cp values exists for side mito H/D = 1.22 art² it rises with the increase in side ratio. But will the increase in angle of attack as the figure reveals the increase in angle of attack as the figure reveals (figure 7e) lowest Cp values occur for side ratio 0 H/D = 2.0. However, from figure 7d it can be observed that at 30° angle of attack the C₁

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A-A Section

was mounted centrally in a horizontal plane in the test section so that the 30mm face was oriented normal to the flow direction (Fig. 1). For angular orientation of the cylinders a graduated disc was used. Mean pressure distribution was recorded at angles of attack varying from 0° to 45° with a step of 5° .

Pressure distributions for each of the cylinders with side ratio of 1.25, 1.5, 1.75 and 2.0 were measured in a similar manner.

The flow velocity in the test section was kept constant at 18.3 m/s (60fps). The Reynolds number based on the side dimension D=30mm was 3.45×10^4 . the turbulence intensity of the tunnel was approximately 0.33%.

The pressure co-efficient is defined as

The drag and lift co-efficients were defined respectively by the following equatins

$$C_{\rm D} = \frac{F_{\rm D}}{\frac{1}{2} \rho A U_{\rm c}^2} \qquad (2)$$

$$C_{L} = \frac{F_{L}}{\frac{1}{2} \circ AU^{2}}$$
(3)

The C_p , C_D and C_L values were determined by numerical integration using Simpson's Rule.

RESULTS AND DISCUSSIONS

Figures 2 to 5 show the effect of angle of attack on mean pressure co-efficients around the rectangular cylinders for side ratios of H/D = 1.25, 1.5, 1.75 and 2.0 respectively. It is observed from these figures that the overall patterns of the Cpdistribution curves for all the rectangular bodies on all the four surfaces are almost similar. However, remarkable variations in C_p-distributions is observed on the windward side (bottom surface) of each cylinder at small angle angle of attack due to change in side dimension. The effect of side ratios on the C_p-distributions for windward side of the rectangular cylinders at angles of attack of 0°, 5°, 10°, 15°, and 30° are shown in figure 6. It is clearly seen from the figure 6(a) that there is rise in pressure with the increase in the side ratio (H/D) of rectangular cylinders at 0° angle of attack. Due to the increase of side ratio of the cylinder, the curvature of shear layer originating from the front surface corner probably decreases which results in the rise of pressure. From figure 6(b) one may observe that there is also rise of pressure as the side ratio vlaue increases. This rise of pressure occurs at a higher rate towards the rear corner of the surface. From the observation of figure 6(c) indication of reatachment of flow towards the rear corner is evident for the cylinder with side ratio of H/D =1.75 and 2.0 at 10° angle of attack. Also the location of the minimum value of C_n shifts towards the front corner with the increase of side ratios of the cylindess. Realtachment of flow towards the rear corner of the bottom surface is prominent at all side ratios of the cylinders for 150 angle of attack as shown in figure 6(d). One may observe that the peak Cp values are shifted towards the front corner with the increase in the side ratio of the rectangular cylinders. At 30° angle of attack it is seen that for side ratio of H/D = 1.75 and 2.0 the peak C_p value is almost at the front corner suggesting separated flow.

The effect of side ratios on C_p distributions on the back surface of cylinder at angles of attack of 0⁰, 10⁰, 20⁰, 30⁰ and 45⁰ are shown in figure 7. It is seen that at 0⁰ angle of attack (figure 7a) minimum C_p values exists for side ratio H/D = 1.25 and it rises with the increase in side ratio. But with the increase in angle of attack as the figure reveals C_p values diminishes and at angle of attack of 45⁰ (figure 7e) lowest C_p values occur for side ratio of H/D = 2.0. However, from figure 7d it can be observed that at 30⁰ angle of attack the C_p distributions for all side ratios are very close.

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Figure 2: Effect of angle attack (α) on Cp-distribution at side ratio (H/D) of 1.25.



Figure 3: Effect of angle of attack on Cp-distribution at side ratio (H/D) of 1.50.



Figure 4: Effect of angle of attack (α) on Cp-distributions at side ratio (H/D) of 1.75.



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Figure 11: variation of drag coefficients with side ratios at angle of attack of 0°.





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The magnitude of the back pressure is determined almost solely by the manner in which the shear layers leave the body and roll up to form discrete vortices. Thus a low base pressure is associated with vortex formation close to the body while a high base pressure (less negative value) is caused by vortex formation further away. The rise in pressure at the back surface of the rectangular body with gradual increase of angle of attack happens due to vortex formation at larger distance from the back side and the fall in pressure with further increase of angle of attack occurs due to vortex formation at smaller distance from the back surface.

The variation of drag-coefficient (CD) for different side ratios (H/D) with angle of attack from 0^o to 45^o is shown in figure 8. It is observed from this figure that for all side ratios the general trend in drag variation is such that with the increase of angle of attack the drag co-efficient falls and becomes minimum in the region of angle of attack 8° to 12° and subsequently with further increase of angle of attack its value rises sharply upto the angle of attack of 45°. Also maximum drag occur at 45° angle of attack for the cylinder with side ratio H/D = 2.0. It is already mentioned that as the angle of attack increases the vortex formation occurs at larger distance thereby creating higher back pressure which is mainly the cause of lower drag with increased value of angle of attack. For further increase of angle of attack vortex formation appear closer to the back surface of the body making the back pressure lower which is mainly the reason of higher drag for the increased value of angle of attack in the higher range.

The variation of lift co-efficient (C_L) with angle of attack for different side ratios is shown in figure 9. The general trend in the variation of lift with angle of attack is similar to that of drag, the high negative lift at small angle of attack is associated with The formation of large enclosed separation bubble on the bottom surface of the cylinder which caused higher local suctions than those on the top surface. As mentioned earlier the reattachment point on the bottom surface shifts towards the front corner with increase in the angle of attack thereby reducing the size of the separation bubble. This results in rise of lift co-efficient for further increase in angle of attack. However, beyond 25° angle of attack no appreciable change in lift occurs with increase in angle of attack for all the cylinders.

The variation of drag co-efficient with side ratios at an angle of attack of 0° is shown in the figure 10. Comparison is also made in this figure with the experimental results presented by P.W.Bearman and Trueman for turbulence intensity of 0.3%. The experimental results provided by P.W.Bearman was for the range of side ratios of 0.2 to 1.25 while the results available for the present study is from 0.5 to 2.0. It can be seen that there is good correlation of the results within the common range of side ratios. Experimental results presented by B.R.Bostok is also compared in this figure for the range of side ratios from 0.4 to 2.0.

It is seen from this figure that drag rises to a maximum value with increase in side ratio upto about H/D = 0.6 and then it rapidly falls with further increase in side ratio. Maximum CD found in the present study is 2.85. The rise in drag co-efficient in the range of small side ratios is probably due to the reduction in the size of the separated wake cavity with increase in side ratio leading to gradual decrease in back pressure and thus the drag coefficient is increased It may be mentioned that due to the rise of side ratio in the smaller range, there appear no reattachment on the side face of the body and hence the distance of the vortex formation is not shifted towards the downstream side; on the contrary due to the increase of side ratio the distance between the back surface and the vortex formation become shorter. According to figure 10 the fall in drag for further rise in side ratio is associated with the direct interference of the downstream edge with the shear layer causing reattachment and thus the vortex formation is delayed further downstream.

CONCLUSIONS

At an angle of attack of 0° , no flow reattachment occurs on the surfaces of each isolated rectangular cylinder. However, the C_p values on the top, bottom and back surfaces rise with the increase in side ratio of the rectangular cylinders.

Flow reattachment commences at smaller angle of attack if the side ratio increases.

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At small angle of attack the C_p value decreases considerably near the front corner of the windward surface of each cylinder and the negative peak value of C_p is shifted towards the front edge with increase in the side ratio.

The minimum drag on the rectangular cylinders occur within 8^o and 12^o angle of attack for all side ratios.

The drag on a rectangular cylinder oriented at 0° angle of attack rises with the increase in side ratio upto about 0.6, then decreases with further increase in the side ratio.

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The nature of the fracture is temperature dependent becares agel andergoes a ductile brittle transition. The Charpy test can be used to demostrate this phenomenon. If the amount of energy needed to fracture the Charpy specimens is plotted against the temperature at which the specimens were instanced, a constructed the type shown in Figure 3 can be constructed the energy needed to fracture the specimens decoded the energy needed to fracture the specimens decoded the energy needed to fracture the specimens decoded the energy needed to fracture the specimens decodes a cliftle behaviourfloads to the the energy of a decover in a ductile manner and below which the material behaves in a brittle manner. The reason for

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excess of 200,000 miles [Jones (1989) and Weiner all. The gas pipe steel used for transmission in exquired to withstand high pressure ranging from 10 1000 psi while the pipe for distribution is required i withstand less than 100 psi. The pipes are generally, the range 24-36 meh in diameter, although gas parts of 42 iech and 48 inch fin diameter are now available.

In order to achieve the economic transmission of large quantities of gas to the rest of the county, it necessary to use high pressure levels - British Ga uses a maximum working pressure of 1000 lbs/m (Whiteman (1978)) The swel pipelines must therefore used specific requirements if the system is to open at safely and effectively.

L. PROPERTIES

The stoel pipes must have adequate

2.1 Strength 2.2 Toughnesser